

EINSTEIN in the Spa

Case studies in thermal baths

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Analysis of present state

Pool areas in Spa's

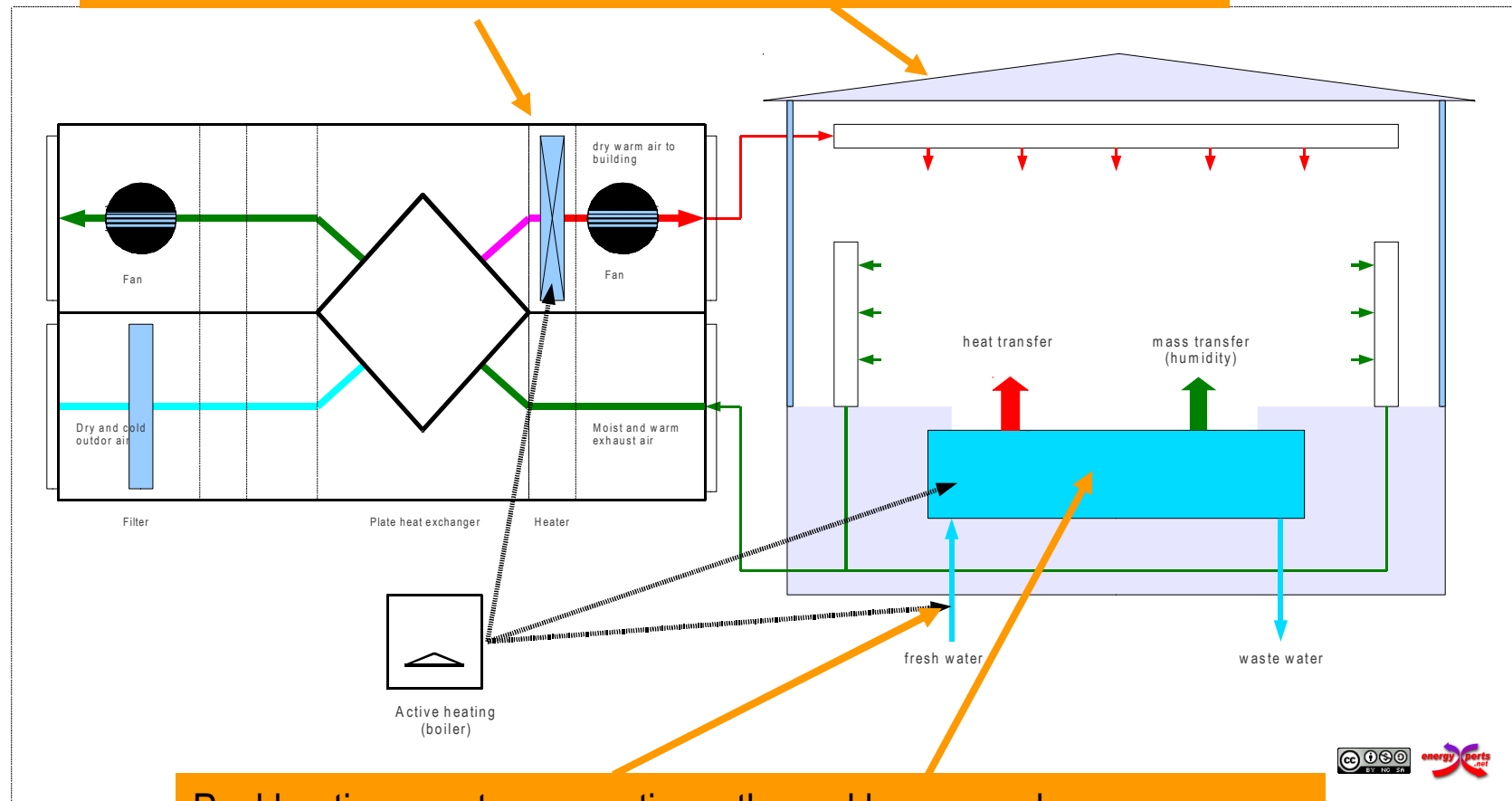
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Energy demand in the Spa (pool areas)

Space heating = air renovation + thermal losses building envelope

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Pool heating = water renovation + thermal losses pool



Typical air handling unit

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Modeling aspects

Straightforward definition of processes might lead to the following:

- Space heating (as building in EINSTEIN)
- Pool heating (as process with circulation: water renovation and maintenance demand: pool losses)

Problem: large amount of heat dissipated from pool into building leads to high energy gains in the building

=> negative maintenance heat demand

$Q_{\text{maintenance}} = \text{Thermal losses building} - \text{internal gains}$

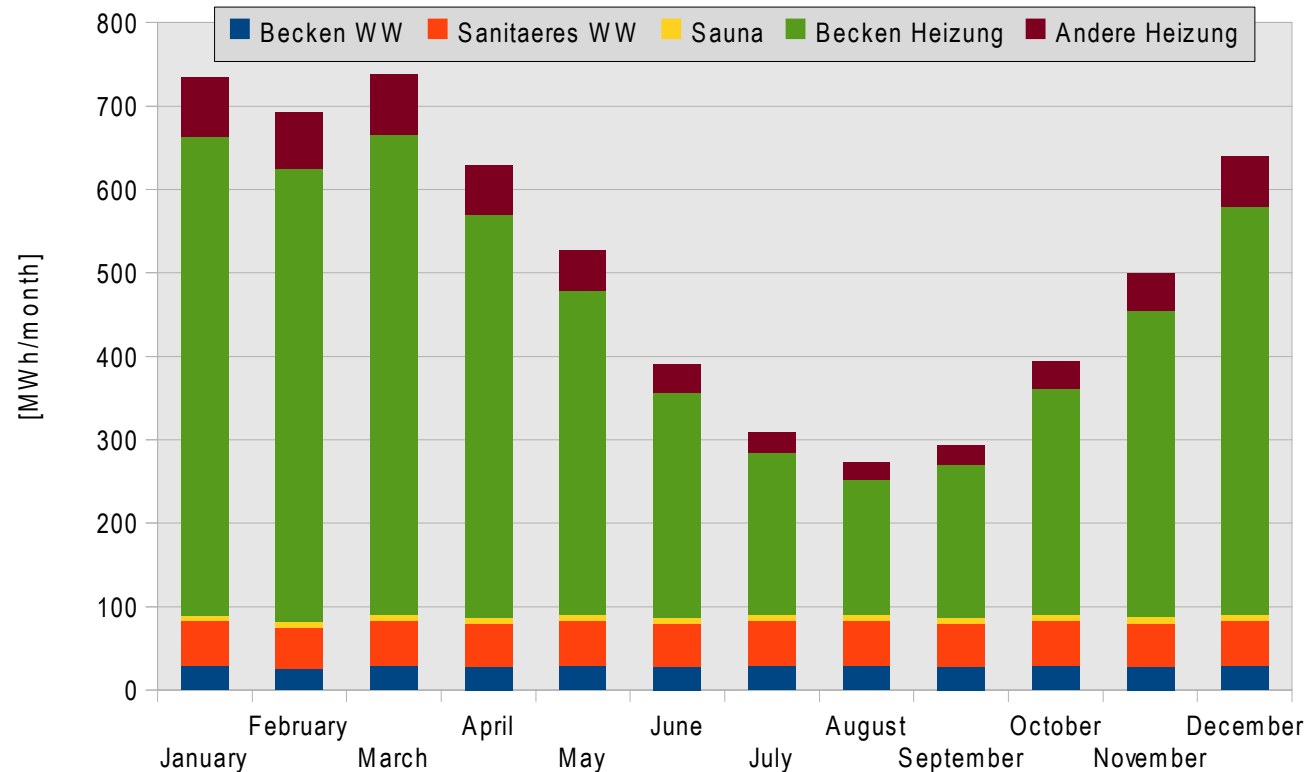
=> direct modeling like this in EINSTEIN not possible, would require “tricks” and manual calculations

Solution: different grouping of energy demands

- Space heating including maintenance of pool temperature (as building in EINSTEIN, all year heating demand)
- Pool heating (water renovation only)

Composition of heat demand (2)

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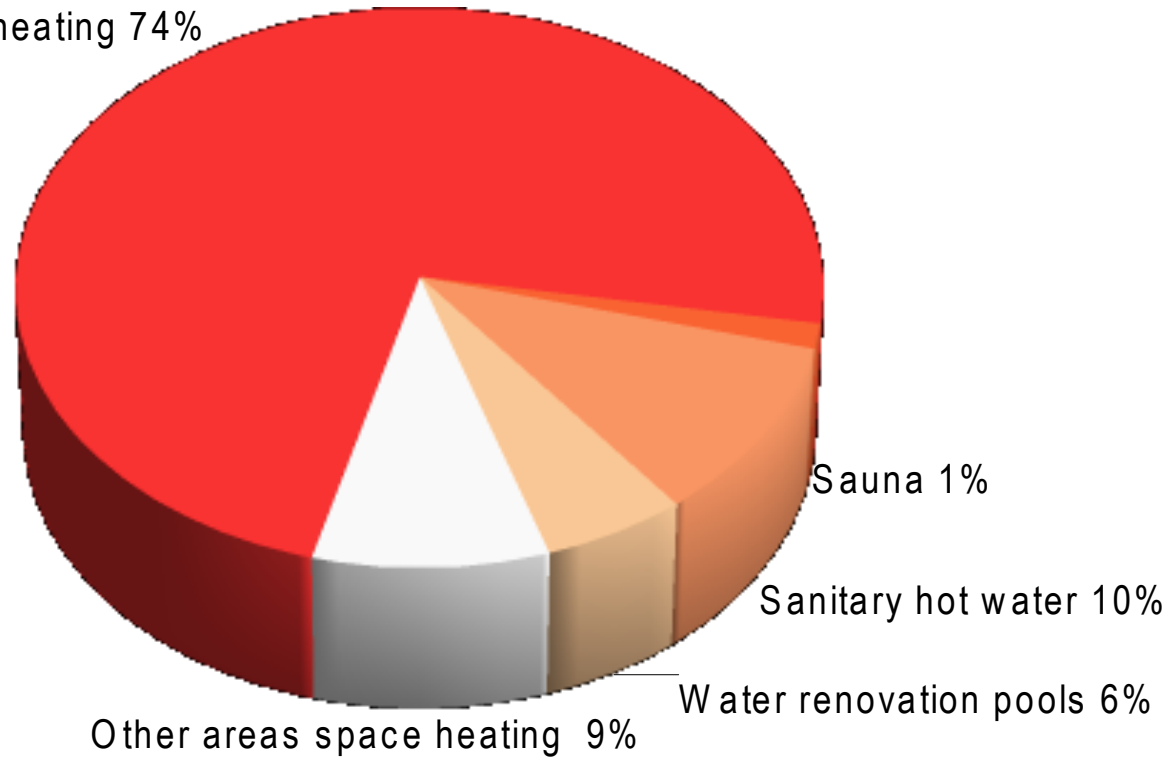
EINSTEIN limitations (restricted user input):

- Ratio of summer to winter heat demand is fixed (f_{scale} oscillates from 0,5 to 1,5)

Composition of heat demand

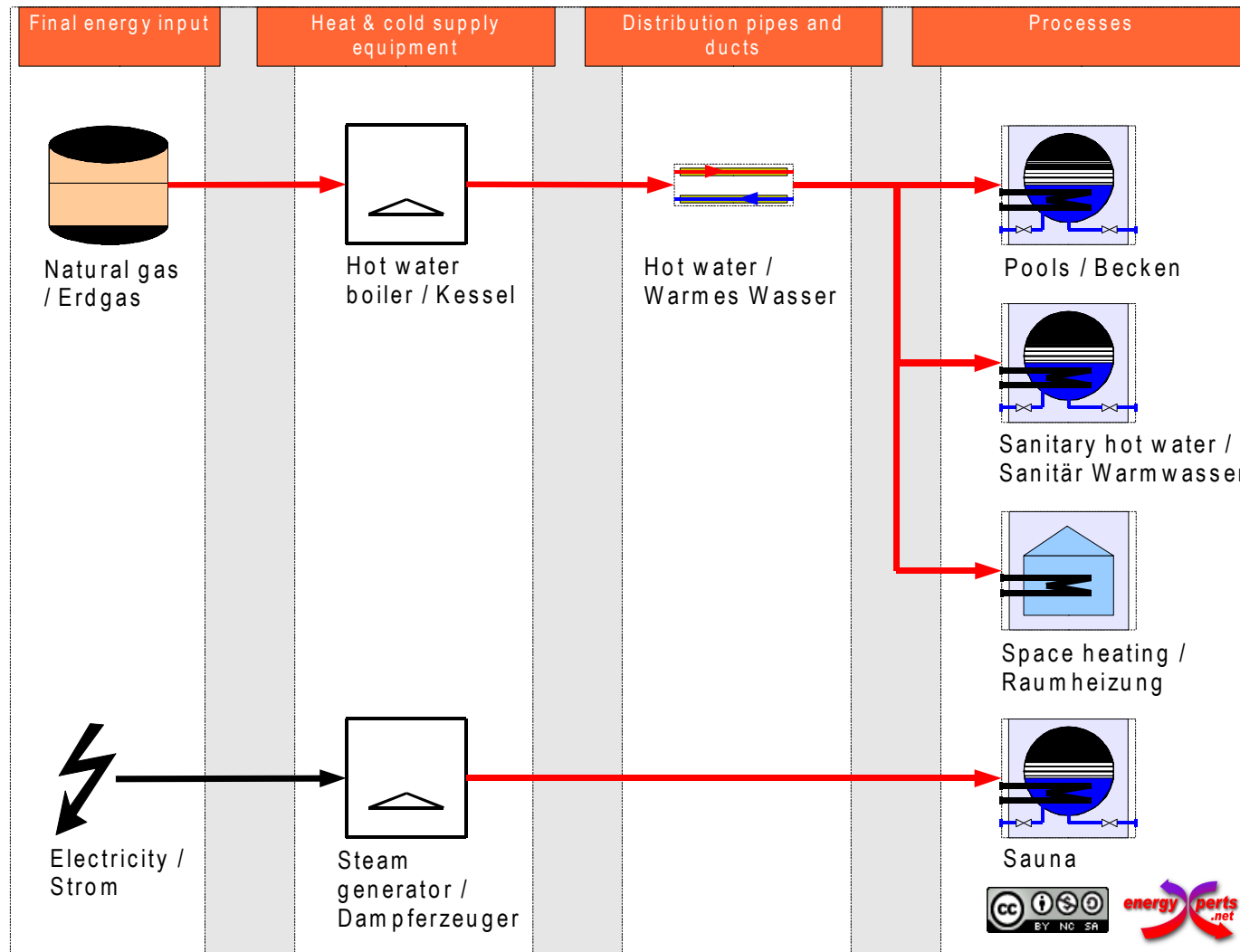
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Pool areas space heating 74%



Heat supply system (present state)

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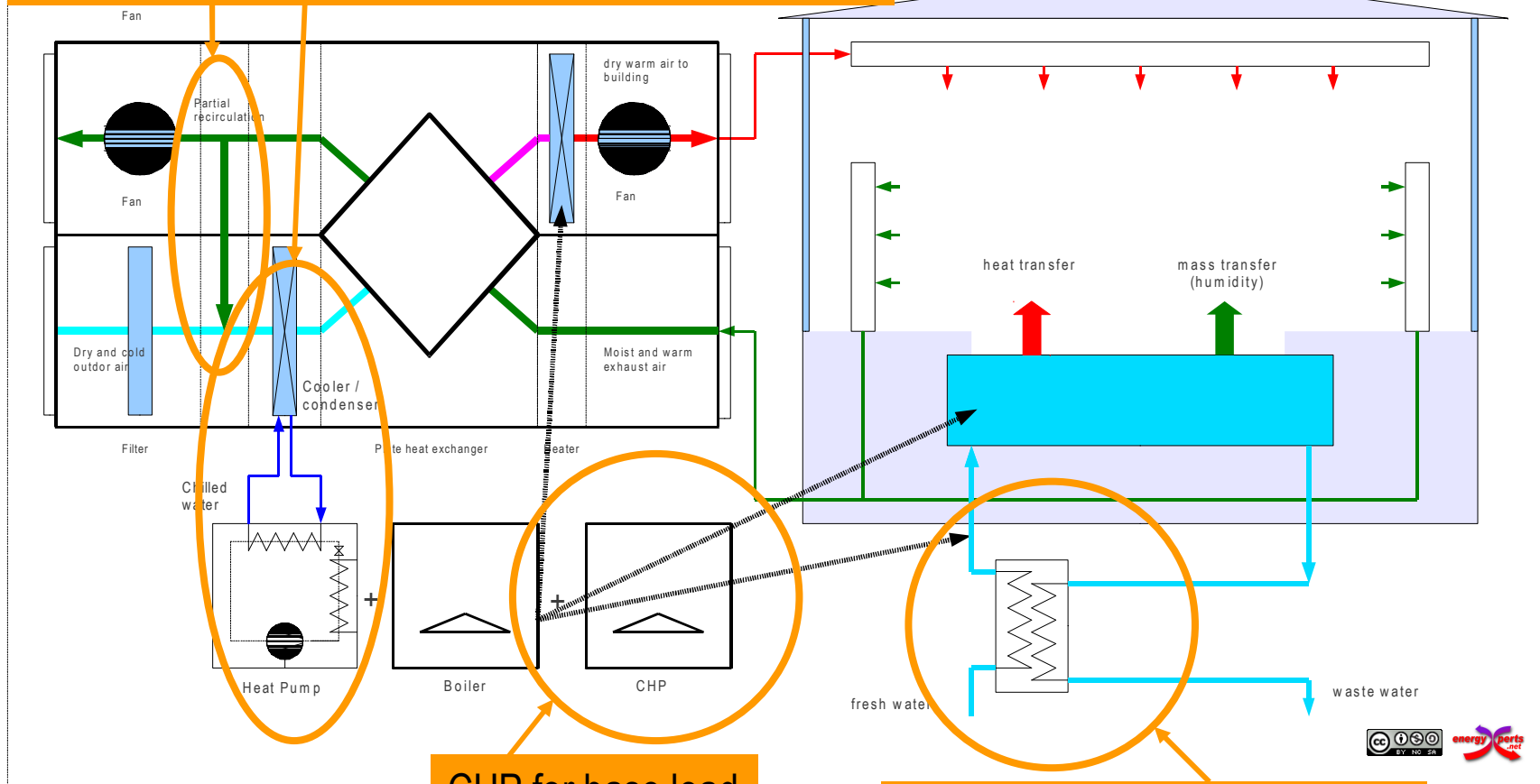


Measures studied

Active dehumidification (heat pump) by cooling below dew point

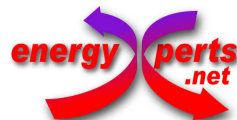
Reduction of air renovation rate (partial recirculation)

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CHP for base load

Heat recovery from waste water



Modeling aspects: active dehumidification

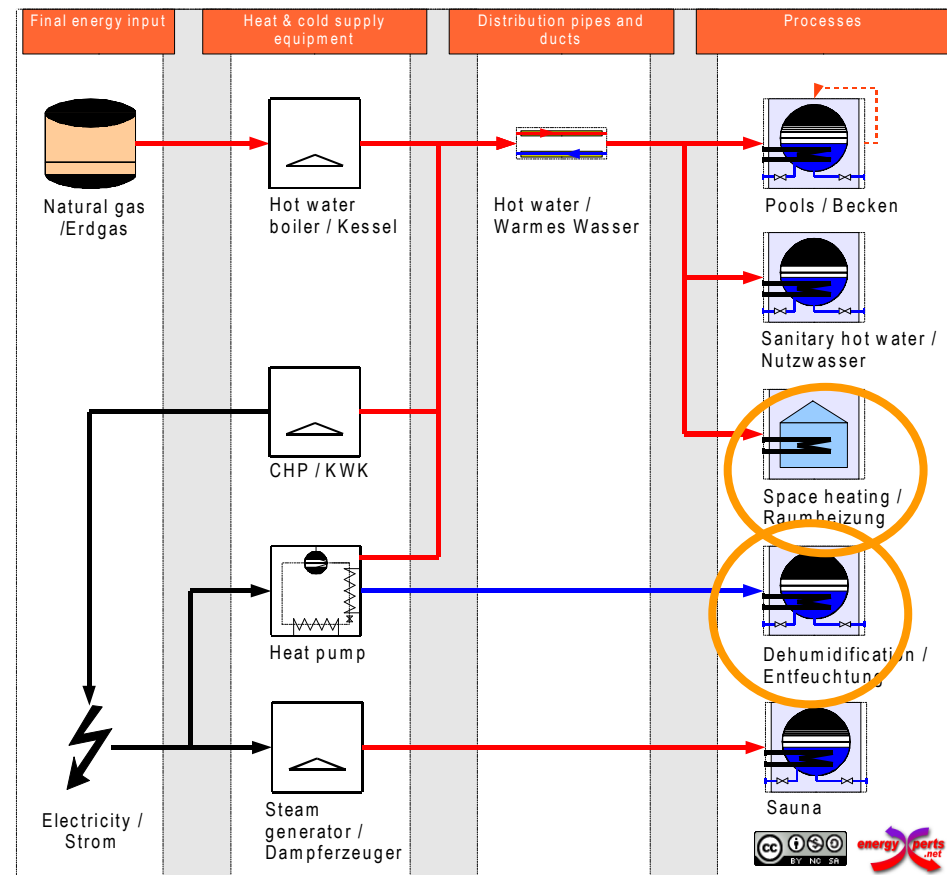
Modification of building heat demand
(now possible in V2.2):

- Reduced air flow rate => reduced circulation heat demand

Introduction of a new process: cooling demand for dehumidification

Hypothesis and manual calculations necessary: no explicit treatment of humidity control in building module

- Hypothesis: Minimum required rate of air renovation
- Conversion of active dehumidification demand into an equivalent cooling demand (cooling power, temperature level)



Modeling aspects: compression heat pump

Heat pump for simultaneous heating and cooling:

- Dummy chiller necessary
- Coupling of cooling side via virtual HX

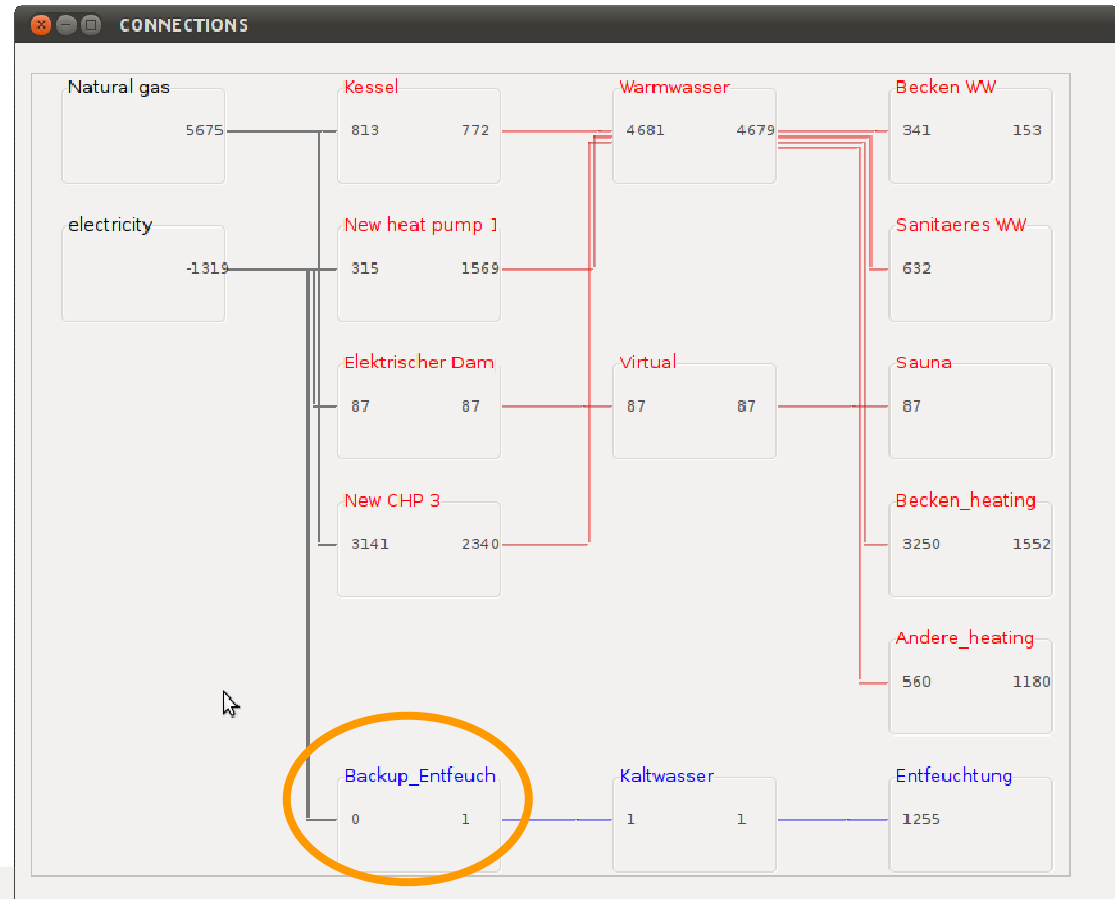
Heat source / sink

Source Sink

Add Delete

Entfeuchtung_Maintenance Inlet temperature Inlet Temperature

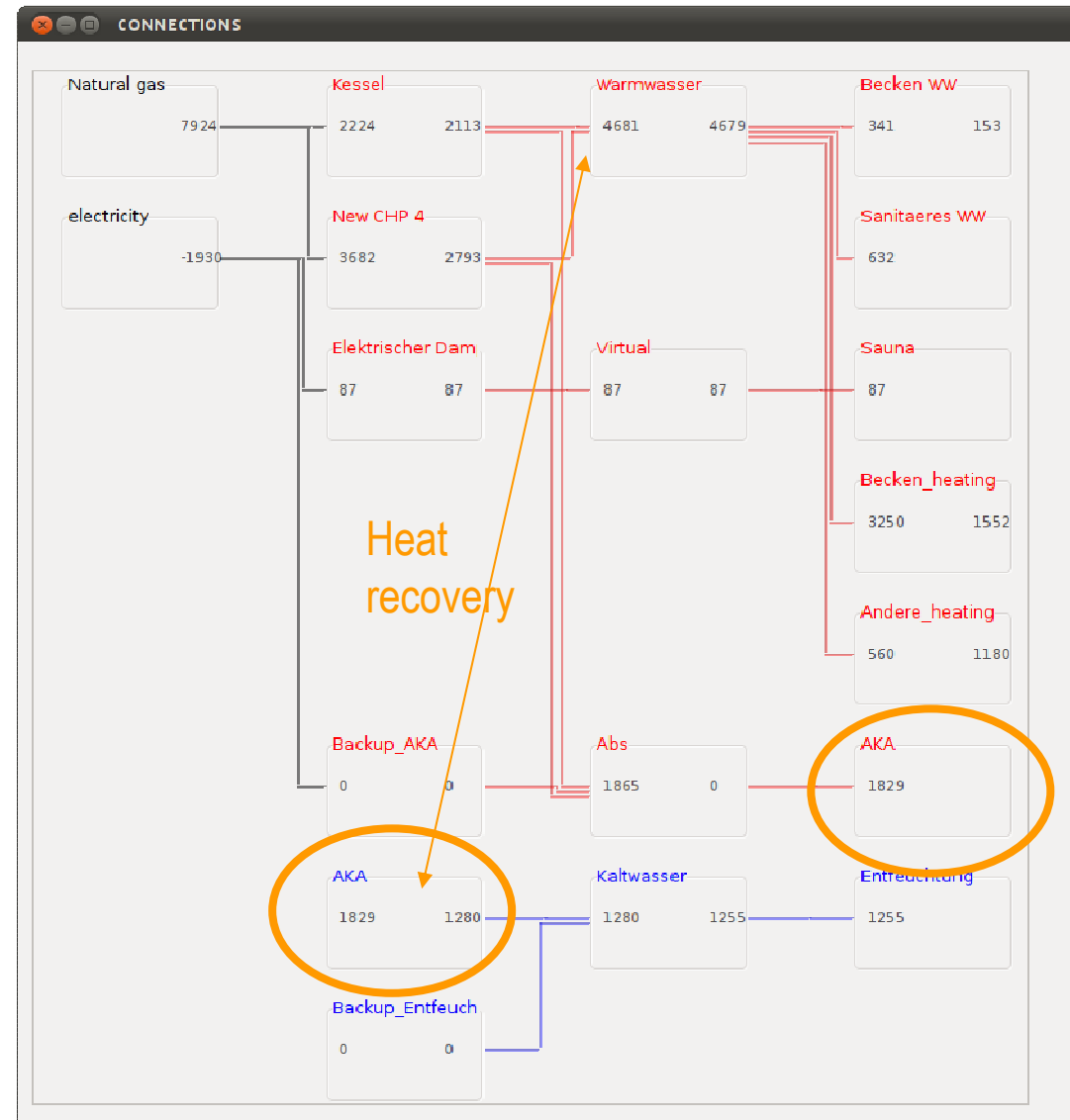
New heat pump 1_Low temperature heating Inlet temperature Inlet Temperature



Modeling aspects: absorption chiller

Absorption chiller for simultaneous heating and cooling:

- Absorption chiller as cooling equipment
- Coupling of heating side via HX (connected to hot water pipe)



Comparison of alternatives

Alternatives studied

KWK:

CHP only (Current project of the company)

KWK + WP (finally proposed alternative):

CHP + active dehumidification (compression heat pump) + heat recovery

KWK + AKA:

Like CHP + WP, but with absorption heat pump

KWK + AKA + ST:

Like CHP + AKA, but additional solar thermal system

KWK2 + AKA:

Like CHP + AKA, but double size of CHP plant

Design process 1: WP vs. AKA

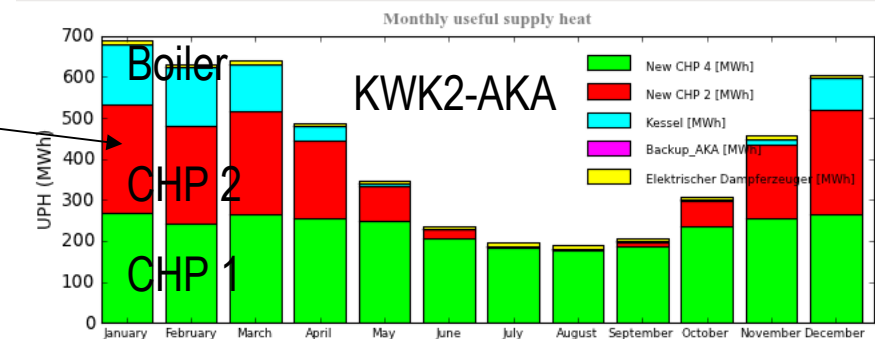
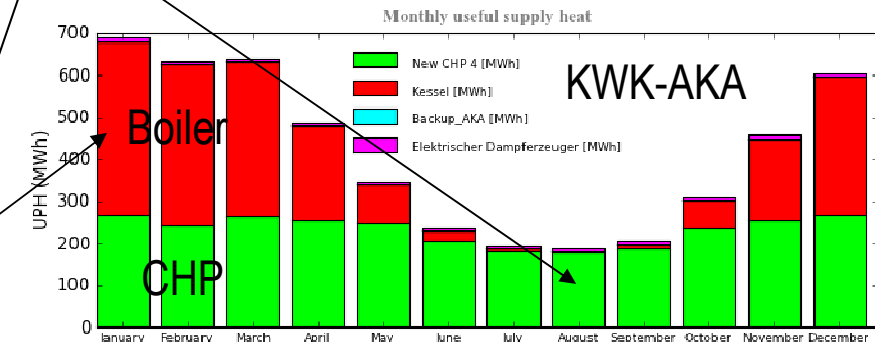
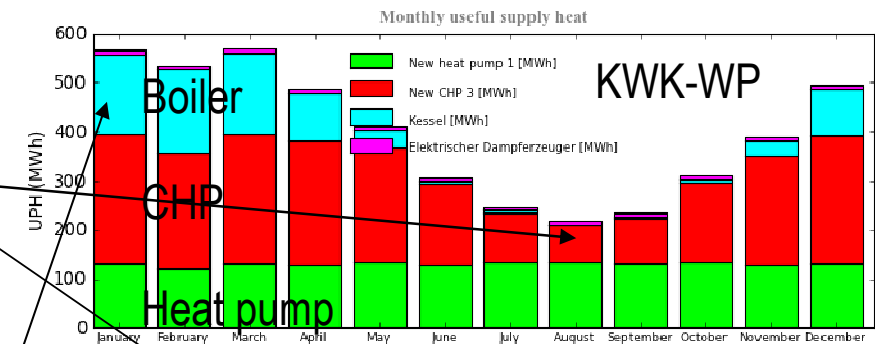
Advantages absorption:

- Higher utilisation of CHP plant in summer

Disadvantage:

- Due to capacity constraint, additional heat demand partially covered by increased input from boiler
- Can be compensated by increased size of CHP

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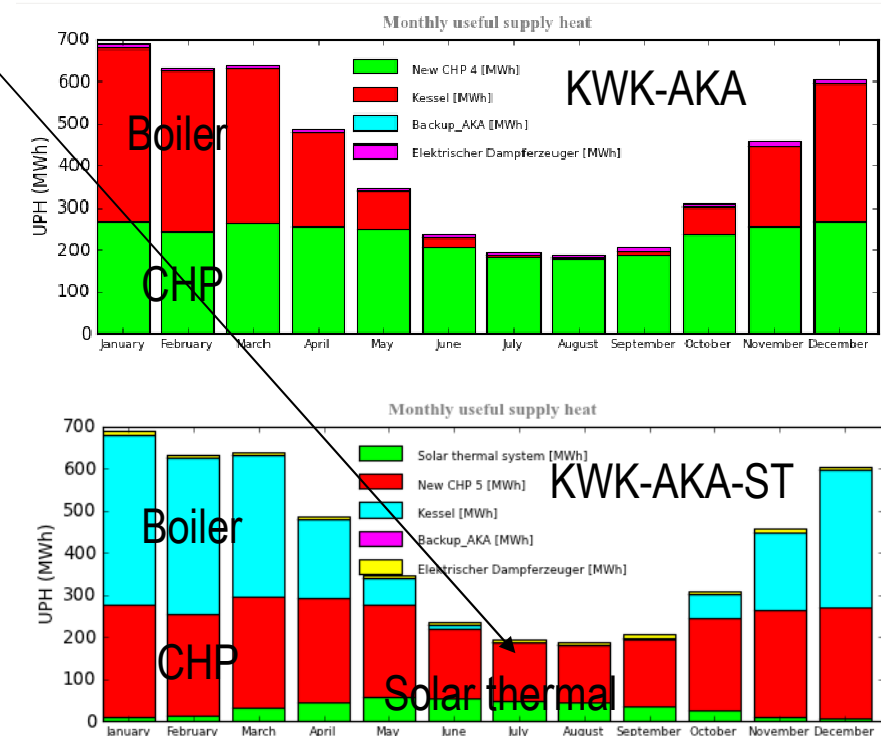


Design process 2: Addition of solar thermal

Problem of combined solutions CHP + solar:

- “Competing” for heat demand in summer

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Alternatives: comparative study

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